

Development of a gas absorption heat pump for residential applications

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POLITECNICO
MILANO 1863





Background / Motivation

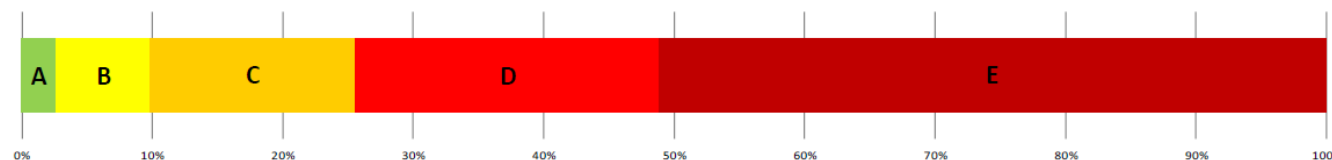


Building market in Europe:

- 75% of the building are inefficient (label D or E);
- Most of these buildings are equipped with radiator with supply temperature > 50 °C;
- In many countries there is a high (about 3) electricity/gas price ratio;
- In many countries the carbon intensity of the electricity is still high, especially in winter.

This implies that about 90% of the energy for space heating is used in building which need:

- High supply temperature and capacity at the same time;
- High efficiency also at high supply temperatures (= no use of inefficient back-up systems);
- A cost-effective solution to decrease CO₂ emissions and reduce the operating costs.



Source: Building Performance Institute Europe

Scope

To develop the thermodynamic cycle and the associated technologies for a gas absorption heat pump for the residential market, able to meet the requirements of the existing buildings.

Two phases:

1. Development of the thermodynamic cycle and introduction of innovative solutions (MGS, Politecnico di Milano, Ariston)
2. Industrialization of a range of products based on the developed concept (Ariston)



Funding

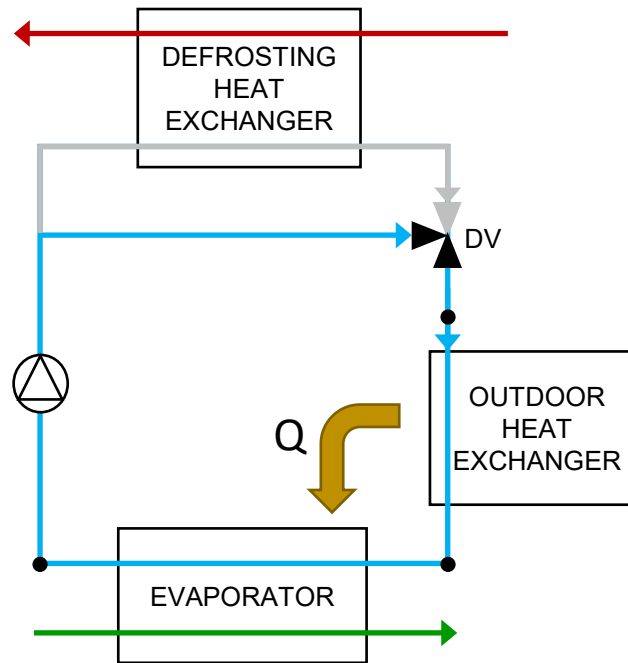
- i-GAP research project, supported under the Call “Agreements for Research and Innovation of the Lombardy Region” co-financed by the ERDF ROP 2014-2020 of the European Commission. Project ID: 241736.
- LombHe@t Project, funded by Lombardy Region under the - Call Hub Ricerca e Innovazione.

In this presentation the main outcomes of the first phase will be described:

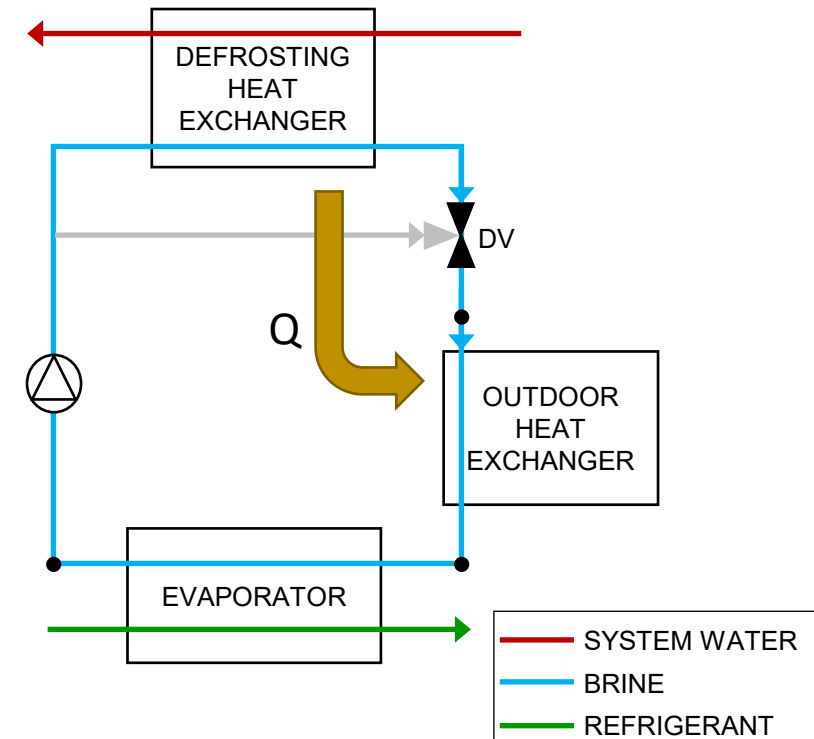
1. Main technological solution implemented in the GAHP to enhance efficiency and flexibility
 - Innovative defrosting process
 - The booster function
 - Use of a two-stage restrictor on the solution branch
2. Seasonal efficiency according to the European Standard EN 12309
3. Operating range

- The GAHP is equipped with an auxiliary brine loop which allows indirect evaporation.
- Normal operation
the brine loop carries the ambient heat to the evaporator.
- Defrosting operation
the brine loop is heated by the system water and provides heat to the finned tube heat exchanger for melting the ice.

NORMAL OPERATION



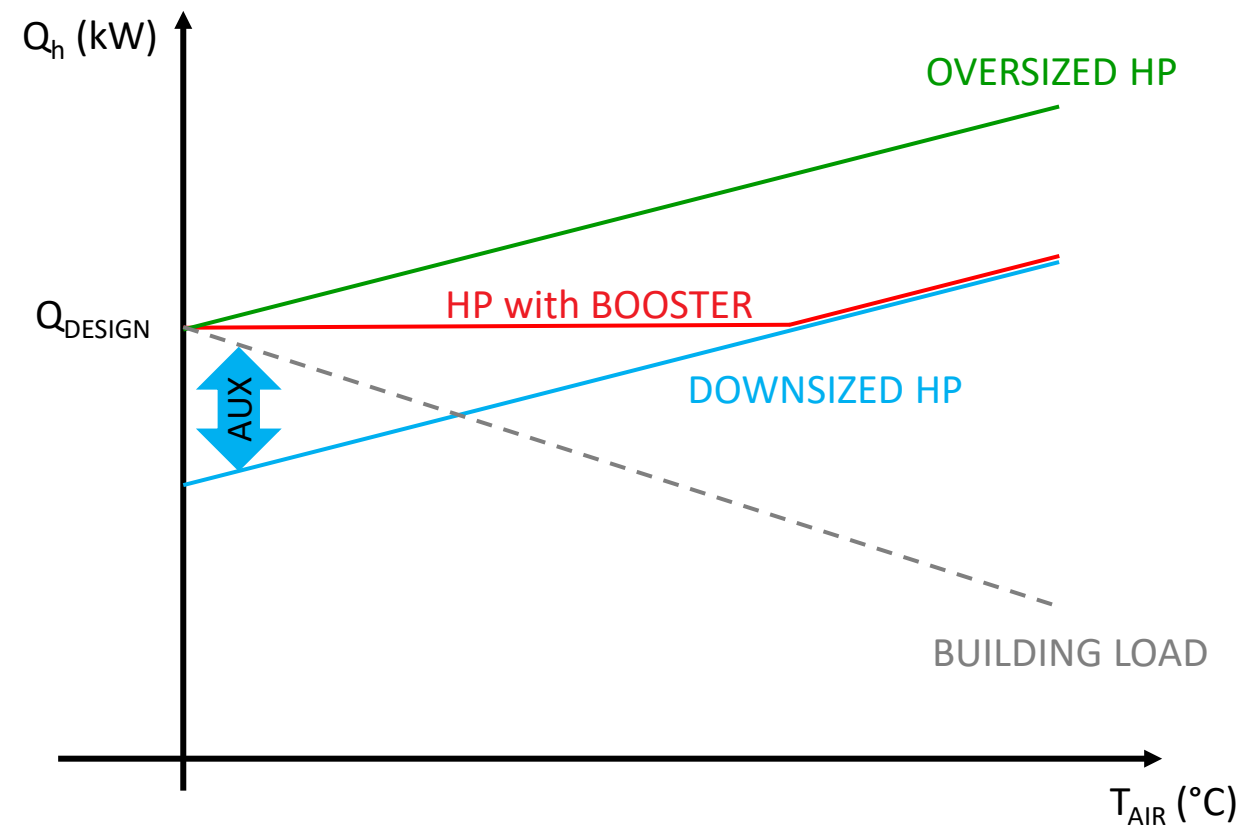
DEFROSTING OPERATION



— SYSTEM WATER
— BRINE
— REFRIGERANT

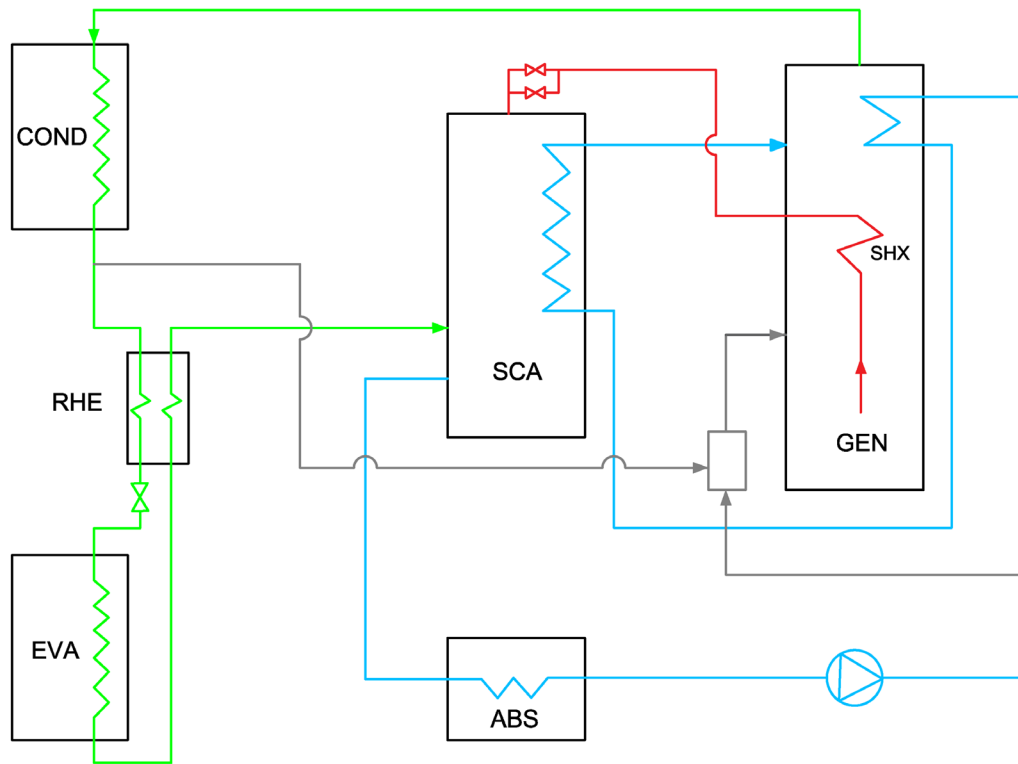
Principle

- Heat pumps usually experience a capacity reduction at high thermal lift i.e., when the capacity demand is high.
- This leads either to:
 - **Oversizing of the unit** (with bad operation at part load conditions);
 - Need for an **auxiliary heating system** (frequently electrical resistor or condensing boiler).
- With the **booster* function** the capacity remains constant by allowing a gas input above the nominal value, while keeping the generator temperature below the maximum value.

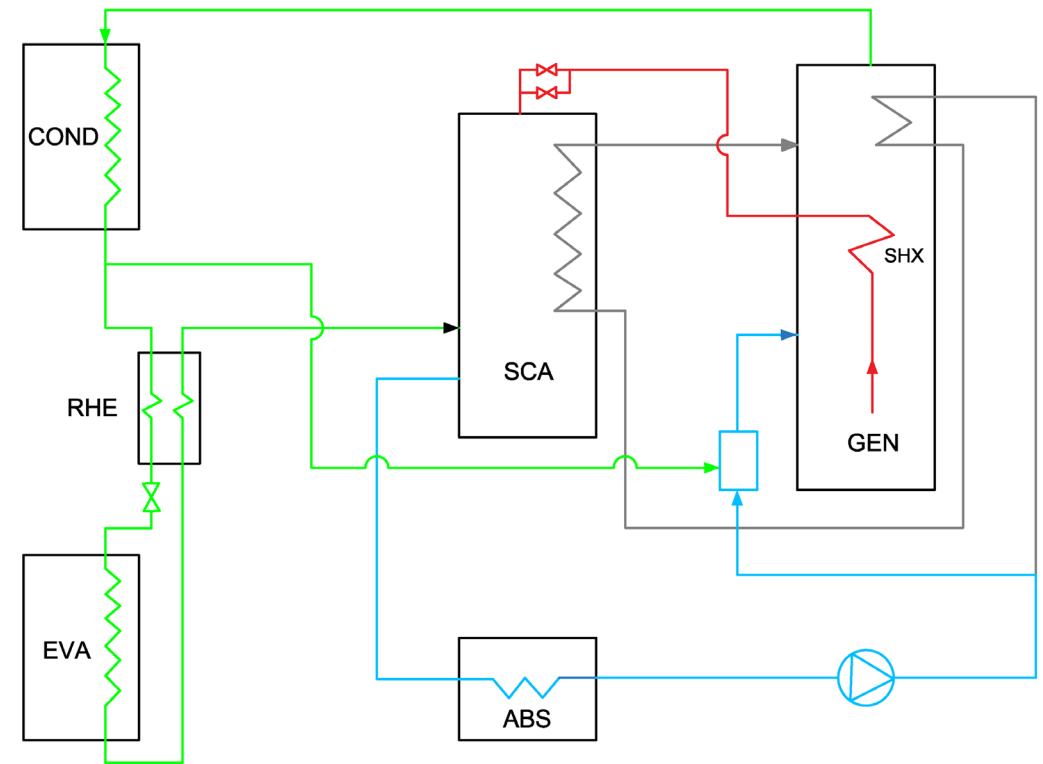


* Marco Guerra, *Absorption heat pump for overfeed generator operating conditions*, Patent application EP2372273B1/ US8950212B2, 2011.

NORMAL OPERATION

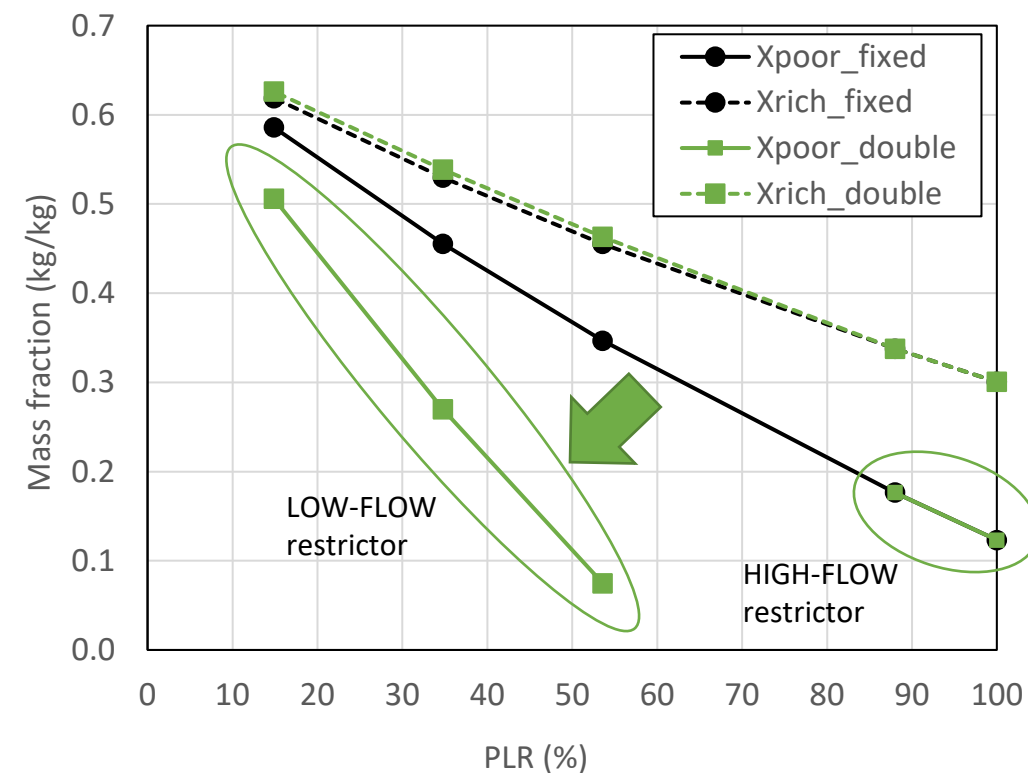


BOOSTER OPERATION



- Part load operation are critical for GAHP with fixed restrictor due to:
 - Reduction of the generator temperature
 - Lower mass fraction spread and high circulation ratio
 - Reduced GAX effect.
- This causes the GAHP to operate under sub-optimal or beyond cut-off conditions.
- This issue can be prevented by adjusting the flow rate of the solution leaving the generator.
- For the developed GAHP a two-stage restrictor has been selected.

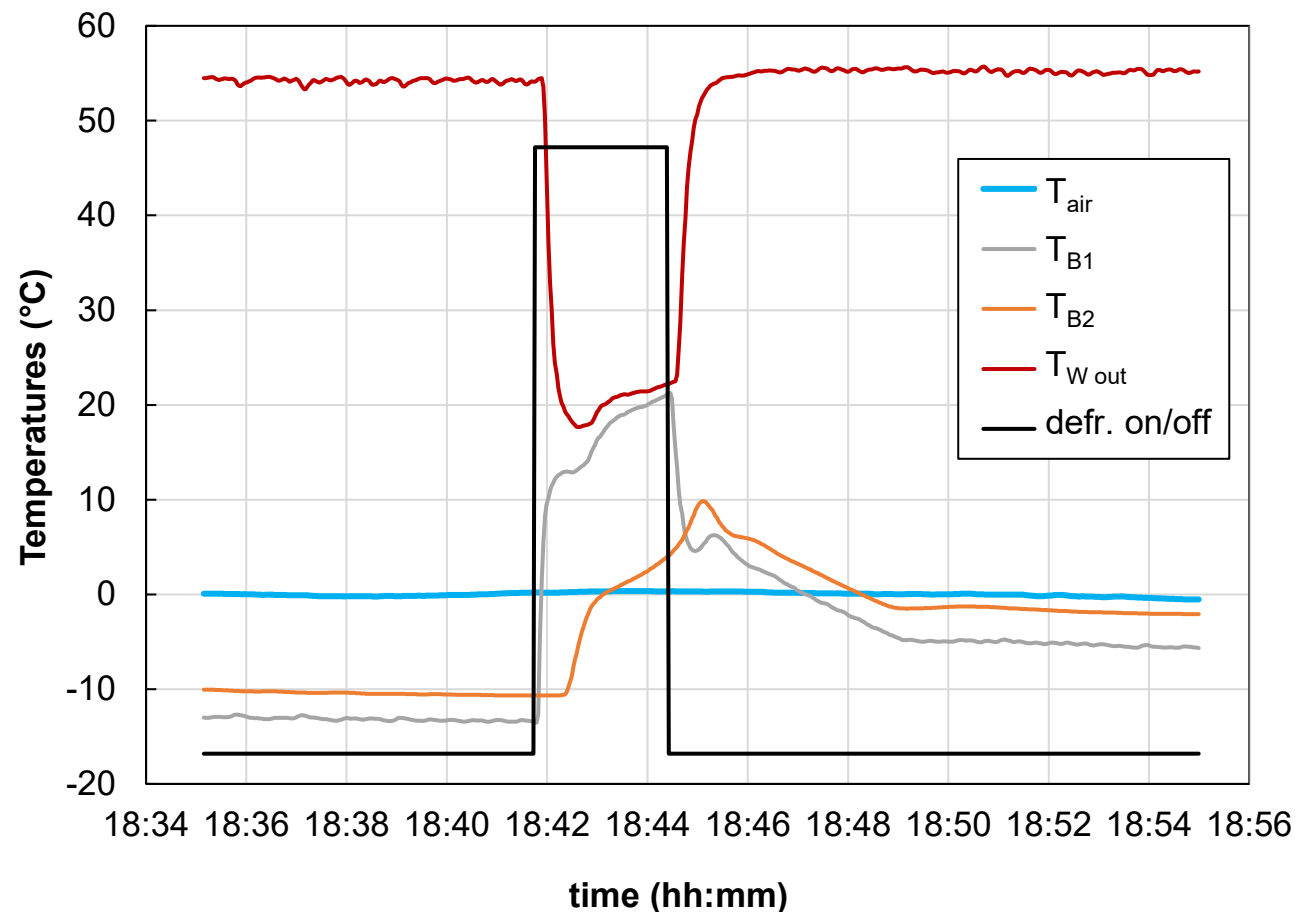
From numerical calculations:



- Defrosting is performed in just about 2 minutes
- The operation has no impact on the thermodynamic cycle



- Even without any additional control, the supply temperature (and the capacity) returns to the set value after about 3 minutes from the beginning of the process.



When activated, the booster function causes:

- A reduction of the generator temperature



Possibility to increase the gas input

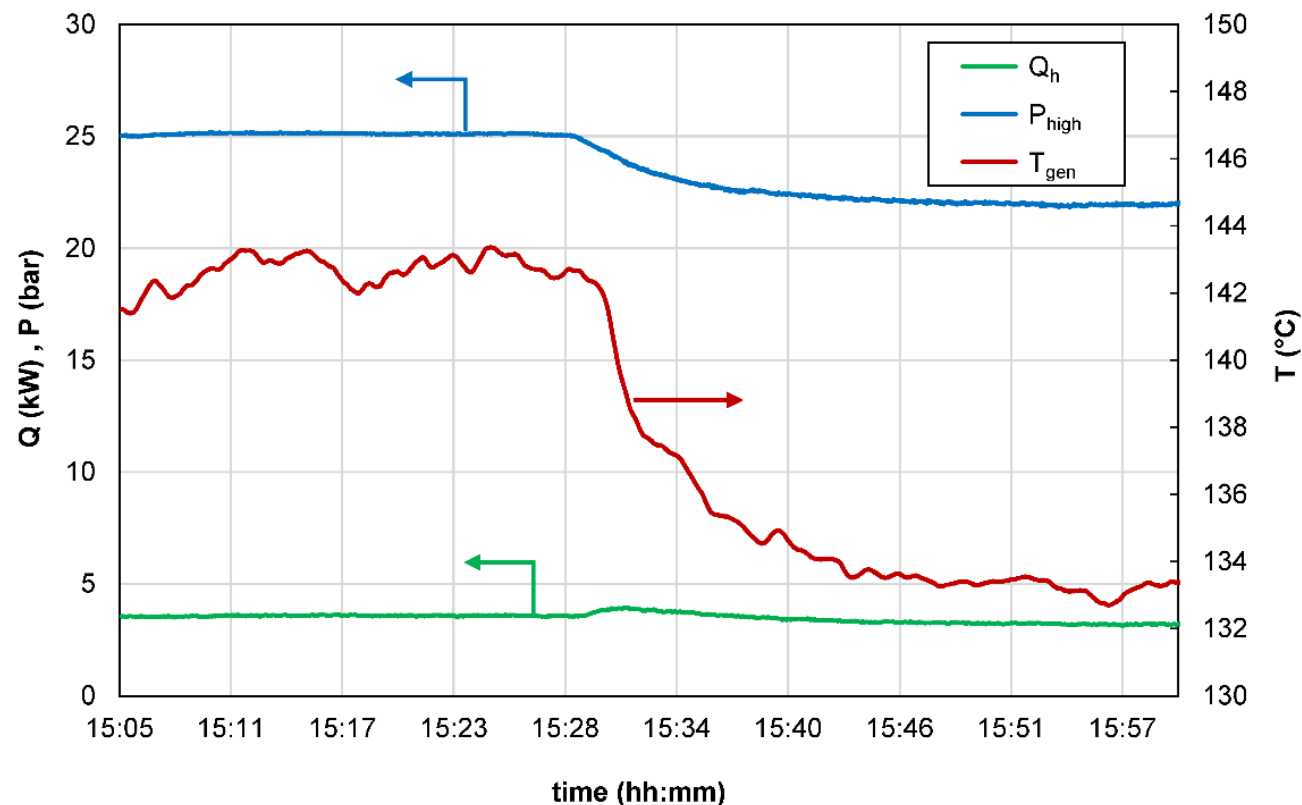
- A reduction of the high pressure



Possibility to increase the water temperature

Test conditions of the example:

- $T_{\text{air}} = 20 \text{ }^\circ\text{C}$; $T_{\text{w}} = 50/65 \text{ }^\circ\text{C}$; $Q_{\text{h}} = 3.6 \text{ kW}$
- Objective: increase both water outlet temperature and heating capacity.



With the two-stage restrictor the poor solution leaving the generator can be set to a low- or to a high-flow

- The low-flow restrictor has been designed to allow stable operation at minimum capacity and low thermal lift
- The high-flow restrictor has been designed to keep the generator temperature below 180 °C at the design conditions

In a range of intermediate conditions both configurations can be used, allowing for the optimization of the performances.



	$T_{air} = 0\text{ °C}$ $T_w = 40/55\text{ °C}$		$T_{air} = 7\text{ °C}$ $T_w = 40/55\text{ °C}$	
	Low flow	High flow	Low flow	High flow
P_{high} (bar)	15.8	16.4	19.2	19.5
P_{low} (bar)	2.7	2.5	4.2	4.3
T_{gen} (°C)	169	144	172	138
Q_{gas} (kW)	5.24	5.47	4.32	4.43
Q_h (kW)	6.33	6.63	6.26	6.27
GUE (-)	1.21	1.21	1.45	1.42

No impact on the Gas Utilization Efficiency (GUE)

Higher efficiency with the Lower flow

The seasonal performances have been calculated based on the European standard EN 12309

Test conditions related to:

- the average climate ($T_{\text{design}} = -10 \text{ }^{\circ}\text{C}$)
- high temperature application ($T_{\text{w nom}} = 55 \text{ }^{\circ}\text{C}$)

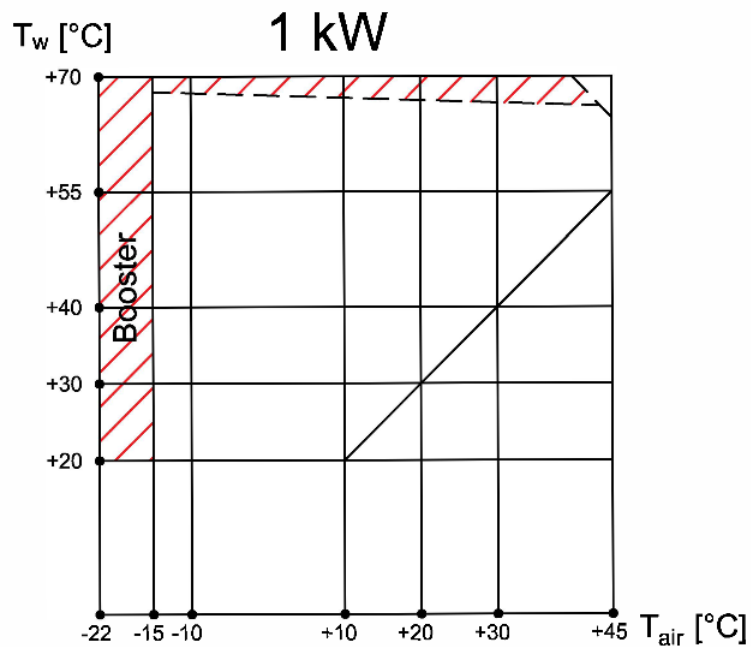
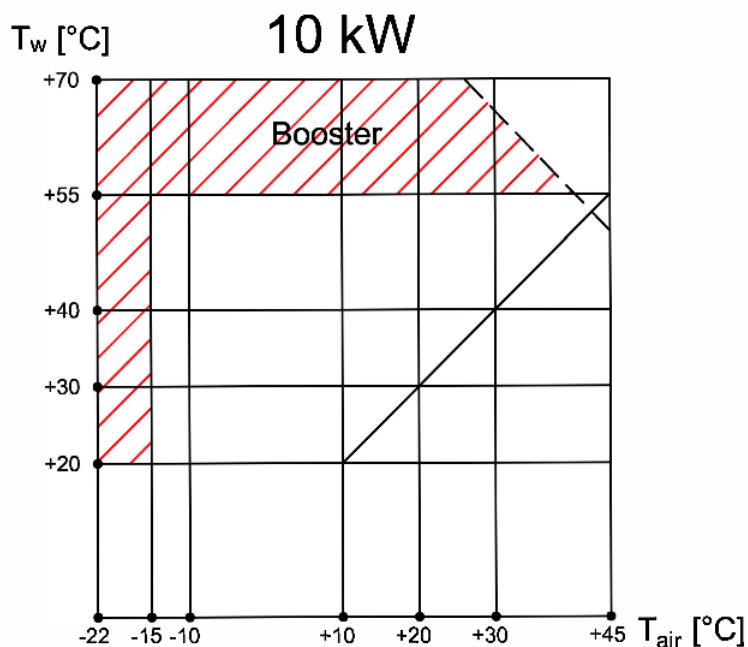
part load ratio	$T_{\text{air}} \text{ (}^{\circ}\text{C)}$	$T_{\text{air WB}} \text{ (}^{\circ}\text{C)}$	$T_{\text{w out}} \text{ (}^{\circ}\text{C)}$	$T_{\text{w in}} \text{ (}^{\circ}\text{C)}$	GUE_{NCV}
100%	-10	-11	55.0	41.4	1.26
88%	-7	-8	52.0	39.4	1.33
54%	2	1	42.0	32.7	1.53
35%	7	6	36.0	28.7	1.61
15%	12	11	30.0	24.7	1.38



Based on the measured values, through interpolation and the use of a bin method the seasonal values are calculated.

Seasonal Gas Utilization Efficiency: **$\text{SGUE}_{\text{NCV}} = 1.50$**
 $\text{SGUE}_{\text{GCV}} = 1.35$

Seasonal Primary Energy Factor: **$\text{SPER} = 1.27$**
 With $f_{\text{GAS}} = 1$ and $f_{\text{EL}} = 2.5$



Thanks to the combined action of the two-stage restrictor and the booster the GAHP:

- Can provide the design capacity in the entire operating range
- Can deliver water up to 70 °C
- Can modulate down to 1/10 of the design capacity

The developed solutions allowed to build a flexible and efficient GAHP suitable for the residential market.

In particular, the GAHP provides at the same time:

- High supply temperature suitable for radiators and DHW production
- Full capacity across the entire working range
- Better efficiency than a condensing boiler even at design conditions
- Lower running costs than a condensing boiler thanks to a seasonal efficiency at least 30% higher

Based on the developed concept, Ariston derived a range of products based on two aeraulic configuration, to address the requirements of the different markets



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